

Plastic nut to be fitted to a component

5 The invention relates to a plastic nut to be fitted to a component having an aperture, which, with a nut piece serving to hold a screw, can be plugged into the aperture and which, in order to bear on one side of the component, is provided with a flange which has a through hole for the screw and, on its side facing the component, has a sealing washer and, in order to bear on the other side of the component, is provided with snap-action hooks which, after insertion, spread out as the
10 screw is tightened in the accommodation hole in the plastic nut and therefore secure the plastic nut to the component.

A plastic nut of this type is illustrated and described in EP 1 143 157 A1. The
15 known plastic nut serves to be inserted in a rectangular hole in a plate, which corresponds to the above-mentioned component. It has a flange which, on one side of the plate, bears against the latter, and a nut piece, which is led through the rectangular hole. Fitted to the nut piece, via a bending zone in each case, are two snap-action books which are oriented obliquely outwards in the direction of the
20 flange and, upon insertion into the rectangular hole, are compressed together until, following complete insertion into the rectangular hole, they spread outwards and therefore secure the plastic nut to the plate. The plastic nut is used in particular for fitting a further module to the component, for which purpose the plastic nut is pushed into the rectangular hole and the module to be fitted is screwed on by
25 means of a screw inserted into the plastic nut. In this case, a flat sealing washer made of rubber fitted to the inner side of the flange ensures a certain amount of sealing in the area of the flange. The idea of using a flat sealing washer in a nut piece that can be inserted into a plate, specifically between a flange of this nut piece and the plate, is also proposed in EP 358 896 B1. However, reliable sealing

of the area on one side of the component with respect to the other side of the component cannot be achieved with the known seals, since leaks can arise, in particular in the area of the screw.

5 The invention is based on the object of configuring the plastic nut in such a way that, by using it, secure sealing is achieved in particular even in the area of the aperture, which is of importance in particular in automobile construction for the purpose of preventing the penetration of water into the inner area of the body-work. According to the invention, this is achieved by the sealing washer having at
10 its outer rim a shoulder which encloses the flange and projects beyond the thickness of the latter and, from its inner rim, merges into a collar which projects into the through hole.

By configuring the sealing washer with the shoulder enclosing the flange and the
15 collar projecting into the through hole, particularly reliable sealing results since, firstly, the shoulder projecting beyond the flange thickness is spread outwards as the screw is tightened and therefore as a further module to be fastened is pressed on, as a result of which the shoulder presses firmly against the further module. Secondly, the collar projecting into the through hole is squeezed together in the
20 process and therefore effects particular sealing around the screw, which means that sealing of all the critical points of the plastic nut can be achieved.

The effectiveness of the sealing in the area of the shoulder can be increased by the shoulder being bevelled on its inner side in such a way that its thickness decreases
25 in the direction away from the flange. Furthermore, the outer rim of the flange can likewise be bevelled to accord with the bevel on the sealing washer. In this case, the oblique outer rim of the flange additionally presses the shoulder of the sealing washer outwards as the screw is tightened, which means that the bevelled shoulder bears on the component with a correspondingly larger area.

A further improvement in the sealing action results if a sealing lip oriented towards the nut piece is integrally moulded on the outer rim of the shoulder, and then presses correspondingly snugly against the flange.

- 5 In the inner region of the sealing washer, too, its sealing action can also be improved, specifically by the sealing washer merging from its side facing away from the flange into a ring that projects into the aperture. This ring is pressed from the inside against the rim of the aperture and effects its own sealing at this point.
- 10 The sealing washer is expediently configured in such a way that it encloses the flange entirely or partially as a covering. In this case, the result is inherent sheathing and encapsulation of the flange, by which means possible leaks between the material of the sealing washer and the flange itself are largely counteracted.
- 15 The effect of the collar may be increased by the latter projecting beyond the flange with an extension. Then, when the screw is tightened, the extension ensures that its material encloses the screw particularly closely and under pressure.

- Inaccuracy in the final distance between flange and component, caused by the
- 20 thickness of the sealing washer, may be eliminated by the region of the flange that is covered by the sealing material being provided with protrusions. The use of these protrusions is possible on both sides of the flange. This is advantageous when a further module is to be screwed onto the plastic nut, which is the general case. The result, therefore, in the region of the sealing washer and possibly in the
 - 25 region of the covering, is zones with a particularly high surface pressure without the sealing material itself being interrupted, the final distance between flange and component and possibly the flange and the further module being determined by the height of the protrusions.

- 30 It is often necessary to bring the plastic nut provided with the sealing washer to bear firmly on the component by tightening the screw screwed into the nut piece,

consideration being given for plate-like components which, for example for tolerance reasons, have different thicknesses. This is done by the sealing washer, apart from the screw, also comprising a compressible spacer which connects the flange and the nut piece in one piece. The spacer, whose height takes into account the thickness of the sealing washer, effects firm contact between the flange on one side of the component and the snap-action hooks on the other side of the component on account of its length being reduced as the screw is tightened, which means that the plastic nut is given a defined, wobble-free position in relation to the component, taking into account the thickness of the sealing washer.

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The accommodation hole for the screw can be formed as a blind hole, which means that, in this region, additional sealing results by means of the blind hole.

The plastic nut described above is excellently suited to be produced by injection moulding, which results in a single-piece component, during whose production the sealing washer can also be moulded on to the flange. For the material of the sealing washer, a resilient material composition which is substantially softer than the material of the actual plastic nut can be selected in this production operation. The plastic nut with its sealing washer moulded on in this way is therefore excellently suited to fabrication in large quantities.

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Exemplary embodiments of the invention are illustrated in the figures, in which:

Figure 1 shows the plastic nut in section according to the line I-I from Figure 3a;

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Figure 2 shows the same plastic nut in a section rotated through 90° according to the line II-II from Figure 3a;

Figure 3a shows the same plastic nut in a section according to the line III-III from Figure 1 with rectilinear spacers;

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Figure 3b shows the same plastic nut in a section according to the line III-III from Figure 1 with rounded spacers;

5 Figure 4 shows the plastic nut in principle in the same configuration but which has a sealing washer modified as compared with Figures 1 and 2;

Figure 5 shows the plastic nut in principle in the same configuration but in which the flange is provided with protrusions;

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Figure 6 shows the plastic nut in a section corresponding to Figure 4, with which nut a further module is connected to the component carrying the plastic nut.

15 Figure 1 shows the plastic nut in section along the line I-I from Figure 3a, which nut is inserted into the component 1 which has the aperture 2 for this purpose. The plastic nut bears with its flange 3 on the upper side 4 of the component 1. The flange 3 is connected to each other via the spacers 5 which can be seen from Figures 2 and 3a, b, which will be discussed in more detail below. The flange 3 is provided with the sealing washer 6, which fills up the area between the flange 3 and the upper side 4 of the component 1. At its outer rim, the sealing washer 6 has the shoulder 7, which is designed to be so high that its upper edge 8 projects beyond the flange 3. At the inner rim of the sealing washer 6, the collar 9 is integrally moulded on the latter, projecting into the through hole 10 of the flange 3.
20 The sealing washer 6 with its shoulder 7 and its collar 9 is configured here as a circular structure and therefore matches the round flange 3. This circular configuration of the shoulder 7 also emerges from Figure 3.

30 The plastic nut according to Figures 1 and 2 also has the nut piece 11 which, in a known manner, contains the two snap-action hooks 12 and 13 which are spread apart when a screw is inserted into the plastic nut (see Figure 7) and, in the proc-

ess, are set behind the component 1 and come into contact with the underside 14 of the component 1. In this way, the plastic nut is firmly held on the component 1 by means of the snap-action hooks 12 and 13 and the flange 3.

- 5 In Figure 2, a section drawn at 90° with respect to the illustration in Figure 2 is reproduced, running in accordance with the section line II-II from Figure 3a. Figure 2 reveals the sealing washer 6 with its shoulder 7 and its collar 9, also showing that the sealing washer 6 comprises the flange 3 and the spacers 5 which are penetrated by the two spacers 5 only along the apertures which can be seen from
10 Figures 3a, b. In the region of the sealing washer 6 beside the shoulder 7, the closed region remains between the spacers 5 and the shoulder 7, so that reliable sealing between flange 3 and component 1 is also ensured over this region.

Figures 3a and b show a section along the line III-III from Figure 1, Figures 3a and b differing only with regard to the configuration of the spacers 5 (see Figure
15 2).

Figure 3a is based on a plastic nut in which the spacers 5 are designed to be extended rectilinearly and, in this form, pass through the sealing washer 6, as Figure
20 2 reveals. In this case, Figure 3a, on the basis of the section along the line III-III from Figure 1, shows the viewer the section surface of the sealing washer 6, which encloses the spacers 5 and, although this can be seen only from Figures 1 and 2, the flange 3. In this case, the through hole 10 is visible to the viewer in Figure 3a. The spacers 5 are shown as sectioned rectilinear walls. Between them,
25 the two snap-action hooks 12 and 13 are indicated by dashed lines. The plastic nut according to Figure 3a is suitable for rectangular apertures in the component.

Figure 3b shows a configuration modified with respect to the illustration of Figure 3a, according to which the spacers 5 are rounded, which means that the plastic nut
30 is substantially suitable for round apertures in the component.

Figure 4 illustrates a plastic screw which, with regard to the configuration of the sealing washer 6, is modified as compared with the embodiment according to Figures 1 to 3a, b. Here, the shoulder 7 enclosing the outer edge 16 of the flange 3 is bevelled on its inner side in such a way that its thickness decreases in the direction
5 away from the flange 3. As a result, the flange 7 is imparted the tendency that, when a screw is tightened and a further module (15, see Figure 6) is fastened, the flange 7 is inclined virtually only outwards and, in the process, bears snugly on the further module without the material of the flange 7 being able to push between this module and the free outer surface of the flange 3. This is undesired since,
10 during the fastening of a further module, the latter is intended to bear snugly on the aforementioned free outer surface of the flange 3, in order that the latter is given a defined position in relation to the plastic nut.

This tendency of the shoulder 7 to kink outwards is further assisted in the embodiment according to Figure 4 in that the outer edge 16 of the flange 3 is likewise bevelled to accord with the bevel of the shoulder 7.
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The sealing washer 6 according to Figure 4 has a still further special feature, namely the sealing lip 17, into which the outer rim of the shoulder 7 emerges.
20 This sealing lip 17 is oriented towards the nut piece 11 and, when the screw is tightened, bears on the component and is squeezed radially away outwards by the latter (see Figure 6) which means that the sealing action of the side of the sealing washer 6 which faces the component 1 is correspondingly intensified.

25 A further special feature of the sealing washer 6 according to the plastic nut according to Figure 4 consists in the sealing washer 6 having, on its side facing away from the flange 3, the ring 18 which projects into the component 1 (see Figure 1), substantially in the axial course of the aperture 2 not shown in Figure 4, and here ensures sealing with respect to the aperture 2.

With regard to the configuration of the sealing washer 6, the plastic nut illustrated in Figure 4 contains a further special feature, which consists in the collar 9 projecting beyond the flange 3 with the extension 19. The extension 19 is bevelled radially inwards which, when a screw is tightened and a further component is pressed on (see Figure 6), leads to the extension 19 being forced away inwards in the direction of the screw and therefore bearing particularly closely on and around the screw. In this case, the extension 19 does not have the tendency to tilt away outwards in the radial direction as well which, for the reasons presented above, would not be desirable, since in this case material from the sealing washer would get between the module to be screwed on and the outer surface of the flange.

It should be pointed out that, of course, all the special features illustrated in conjunction with Figure 4 as compared with the embodiment according to Figure 1 can additionally be applied on their own to the embodiment according to Figure 1.

The embodiment of the plastic nut according to Figure 5 is again based on the basic configuration of the plastic nut according to Figures 1 to 3a, b but again with a modification of the sealing washer 6 and of the flange 3. Here, the sealing washer 6 forms a complete covering over the flange 3, which here extends as a cover 20 over the flange 3. This achieves the situation where the flange 3 is sealed off from all sides. The flange 3 itself is here provided with protrusions 21, specifically on its upper side and its underside, which extend partially into the material of the sealing washer 6 and the cover 20, that is to say into the covering. Because of the arrangement of the protrusions 21, when a screw is tightened and a further module is pressed on (see Figure 6), the result is a high surface pressure and pressing of the material of the sealing washer 6 away over the protrusions 21, of which material in each case only a thin film remains for each protrusion 21, but this is sufficient for a seal to continue to exist. During this procedure, that side of the flange 3 which faces the component 1 bears with its protrusions 21, apart from the remaining film, on this side of the component 1, which correspondingly also takes place on that side of the flange 3 which faces away from the component 1 as

a result of a further module being pressed on, so that ultimately the further module is held at a defined distance, given by the thickness of the flange 3 and the height of the protrusions 21, from the component 1 with the screw tightened.

- 5 In Figure 5, the nut piece in 11 is illustrated with a special feature. Here, with regard to its threaded hole 22, the nut piece 11 is closed off by the base 23, which therefore means that the threaded hole 22 is formed as a blind hole in the embodiment according to Figure 5.
- 10 In Figure 6, the plastic nut, which substantially corresponds to that according to Figure 4, is illustrated in interaction with a component 1 and a further module 15 to be fitted thereto, for which purpose the screw 28 is inserted into the through hole 10 (see Figure 4) and the threaded hole 22 and is tightened, the screw head 29 bearing on the module 15 and the latter bearing firmly on the component 1
- 15 with the interposition of the flange 3. The snap-action hooks 12 and 13 are spread apart in this case and bear on the underside 14 of the component 1. In this case, the sealing washer 6 is compressed highly and its shoulder 7 is forced away outwards, therefore bearing snugly on that side of the module 15 which faces the shoulder 7 and additionally effects sealing here. In a corresponding way, the
- 20 sealing lip 17 (see Figure 4) has also been placed flat on the component 1. Furthermore, the ring 18 projects into the aperture 2 (see Figure 1) of the component 1 and additionally seals off the latter. Finally, the collar 9 with its extension 19 has the effect of enclosing the thread-free neck of the screw 28, around which the collar 9 and the extension 19 are squeezed, and therefore also seals off the screw
- 25 28 in relation to the space assumed by the nut piece 11. Overall, there is now particularly effective sealing of the region that accommodates the screw head 29 and the region occupied by the nut piece 11, which also applies in the case in which any pull-off forces act on the module 15 in relation to the component 1. In this case, if the tendency arises for the flange 3 together with the module 15 to be
- 30 moved slightly away from the component 1, then this enlargement of the space between flange 3 and component 1 which, according to Figure 6, is defined by the

compressed sealing washer 6, will be loosened somewhat, but this can be compensated for in particular by the shoulder 7 and the sealing lip 17, which follow resiliently in this case. In the process, in the region around the thread-free neck of the screw 28, nothing changes in the sealing relationships, so that the sealing effected here by the collar 9 and the extension 19 remains unaffected by the process outlined above. Overall, therefore, even under special loadings which act on the module 15, there is particularly reliable sealing.